US009080446B2

(12) **United States Patent**
**Heusler et al.**

(10) **Patent No.: US 9,080,446 B2**
(45) **Date of Patent: Jul. 14, 2015**

(54) **SCROLL COMPRESSOR WITH CAPTURED THRUST WASHER**

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( * ) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 267 days.

(21) Appl. No.: **13/428,165**

(22) Filed: **Mar. 23, 2012**

(65) **Prior Publication Data**

US 2013/0251574 A1 Sep. 26, 2013

(51) **Int. Cl.**

| | |
|---|---|
| ***F01C 1/063*** | (2006.01) |
| ***F01C 17/06*** | (2006.01) |
| ***F04C 23/00*** | (2006.01) |
| ***F04C 18/02*** | (2006.01) |
| *F04C 21/00* | (2006.01) |
| *F04C 29/12* | (2006.01) |

(52) **U.S. Cl.**

CPC ........... ***F01C 17/066*** (2013.01); ***F04C 18/0215*** (2013.01); ***F04C 23/008*** (2013.01); *F04C 21/007* (2013.01); *F04C 29/126* (2013.01); *F04C 2230/603* (2013.01); *F04C 2240/56* (2013.01); *Y10T 29/4924* (2015.01)

(58) **Field of Classification Search**

CPC . F04C 23/008; F04C 18/0215; F04C 2270/72
USPC .............................. 418/55.1, 55.3, 55.4, 55.5
See application file for complete search history.

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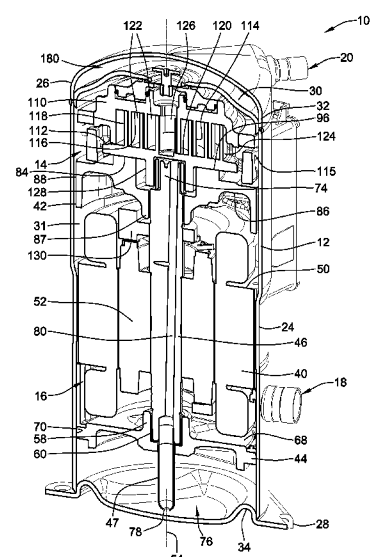
(57) **ABSTRACT**

A load transmittal apparatus transfers an axial load to a thrust surface during operation of a scroll compressor.

**8 Claims, 15 Drawing Sheets**



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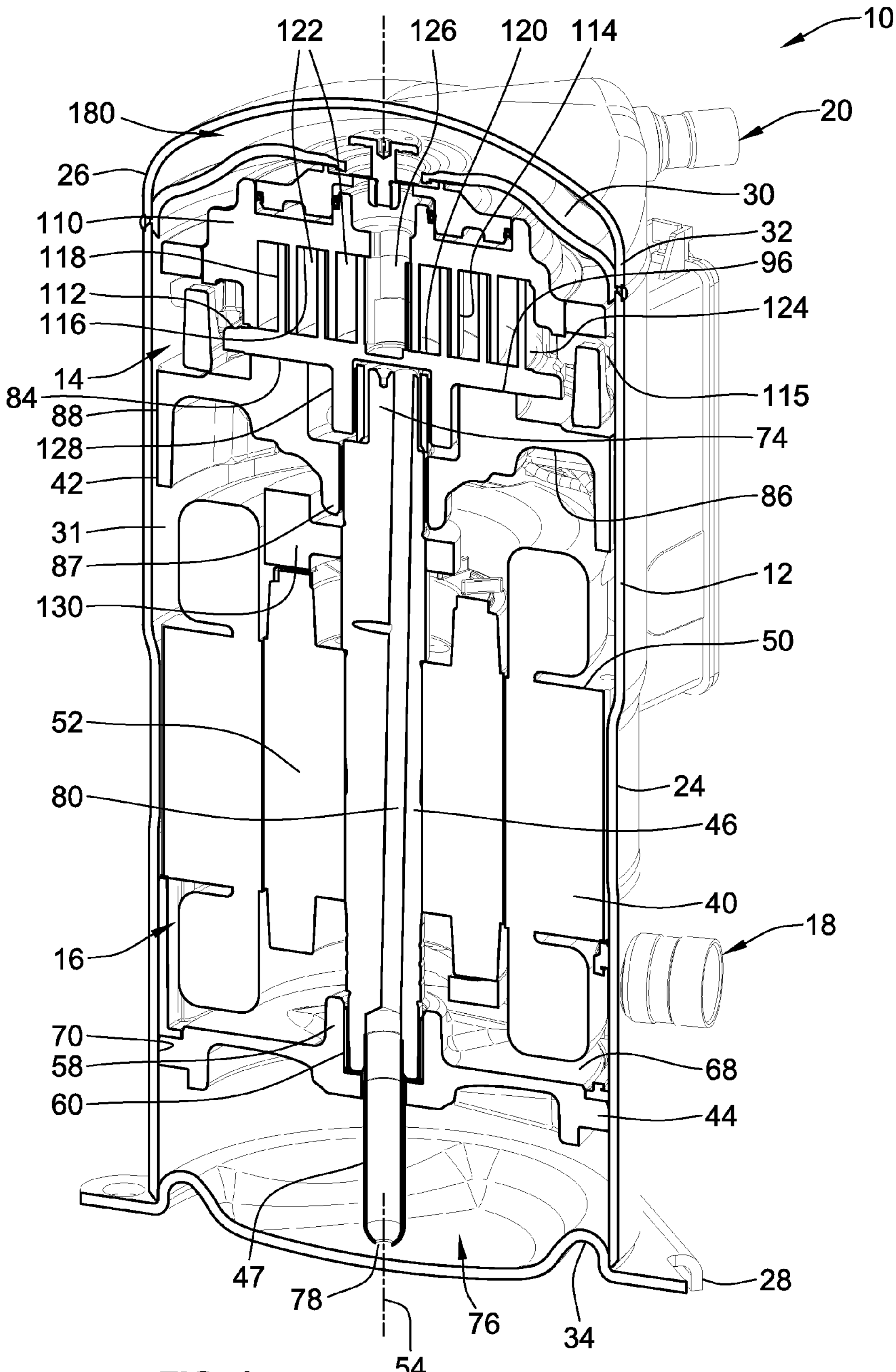
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FIG. 1

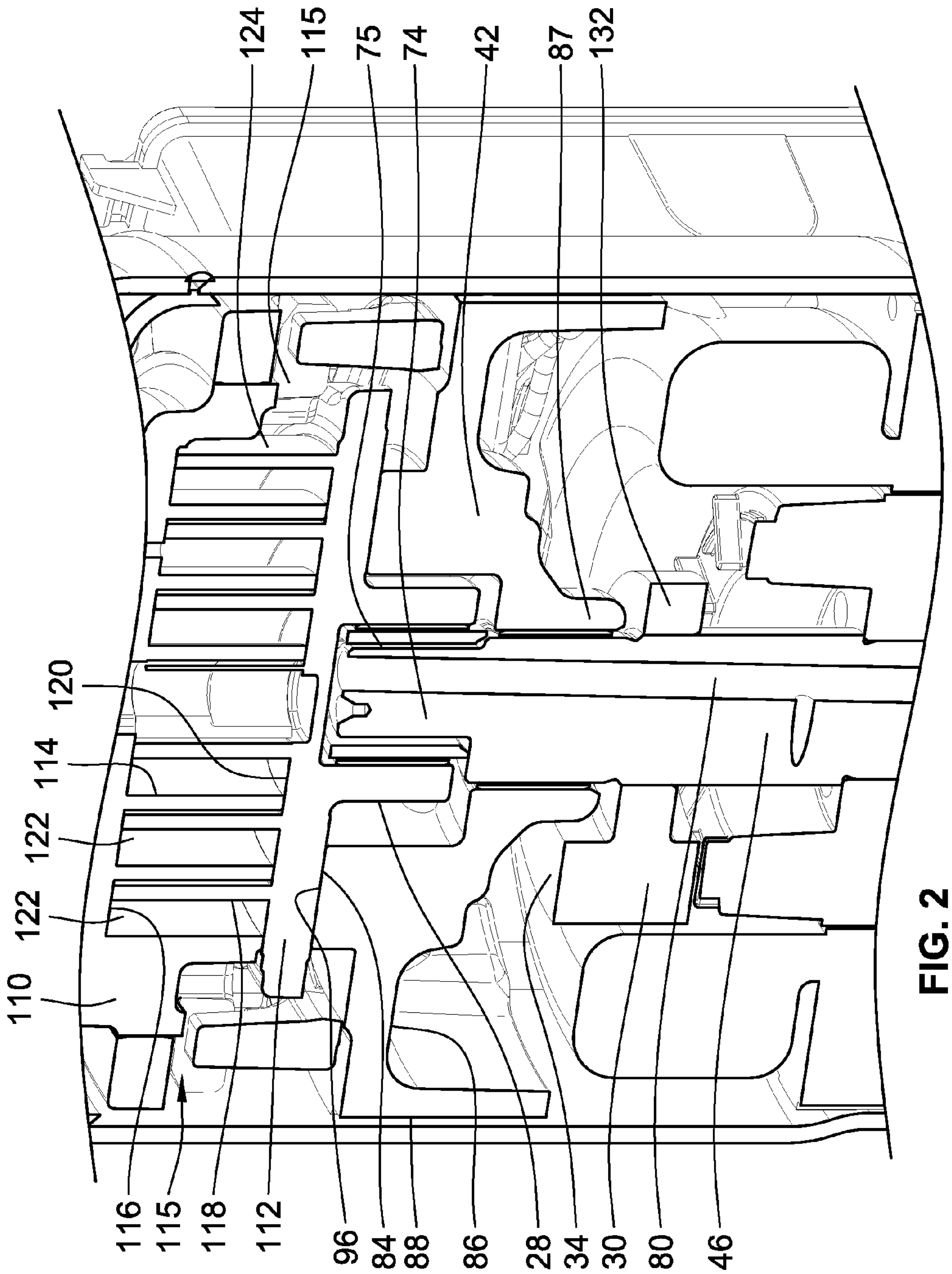
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FIG. 2

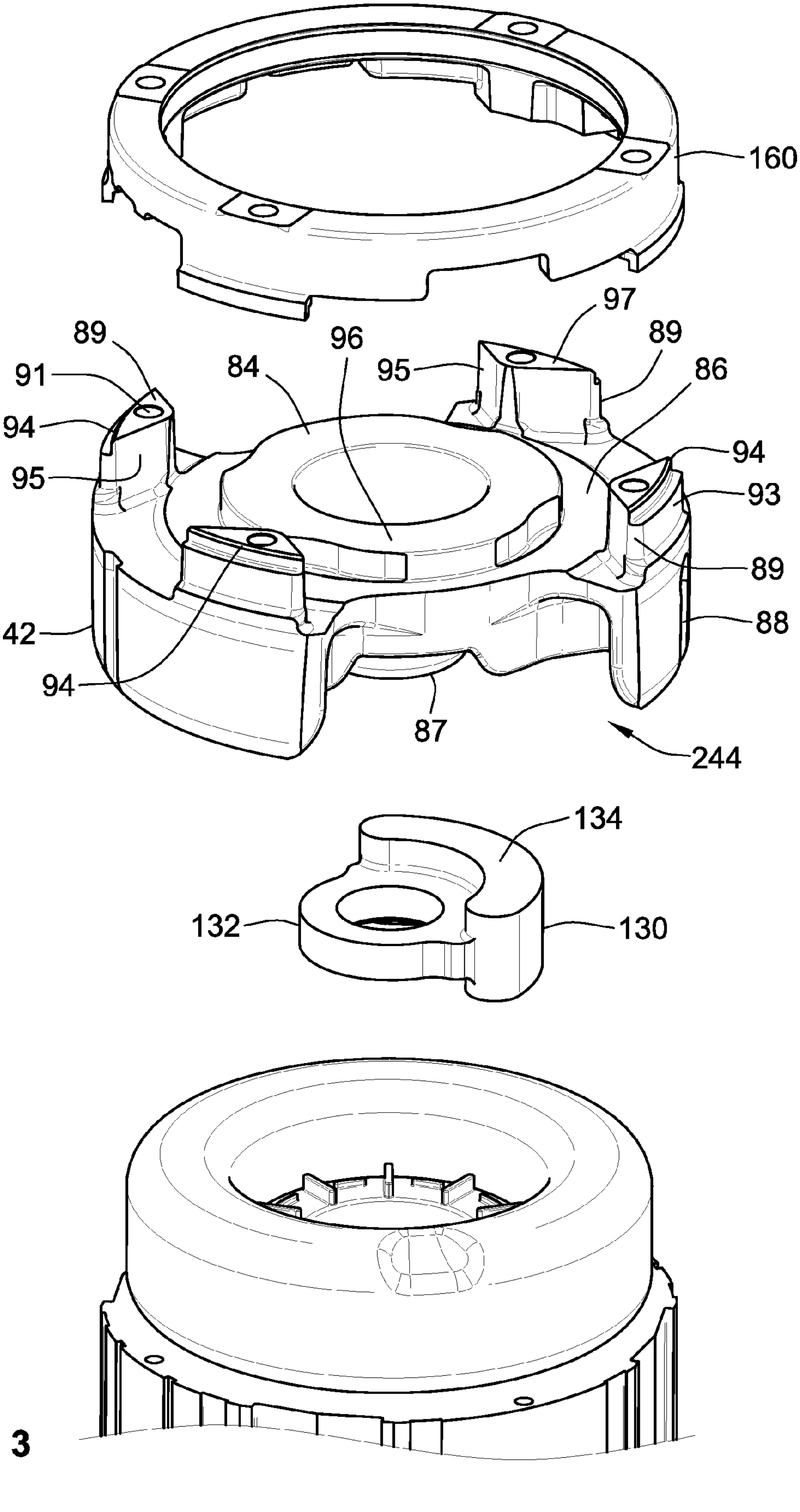


FIG. 3

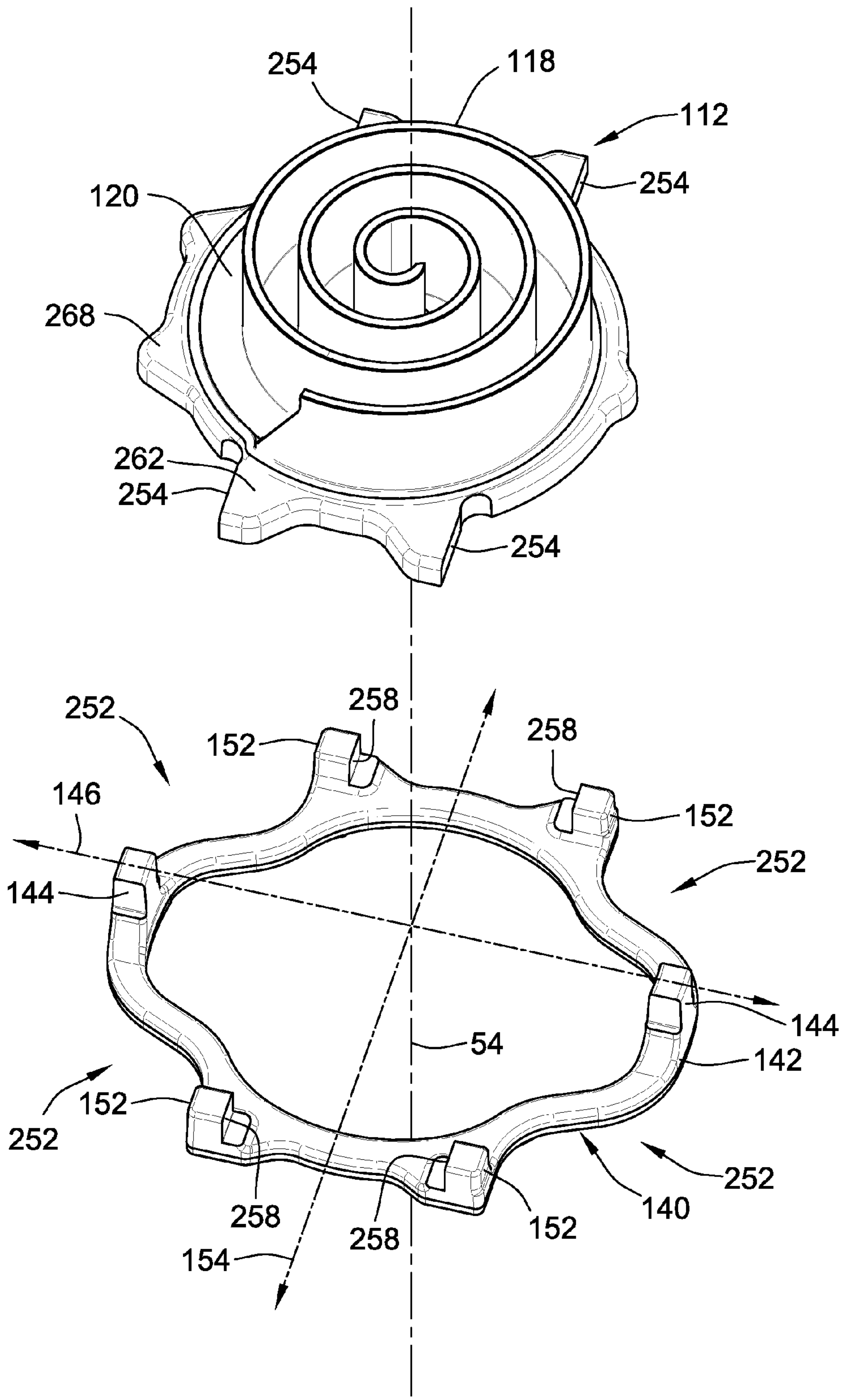
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FIG. 4

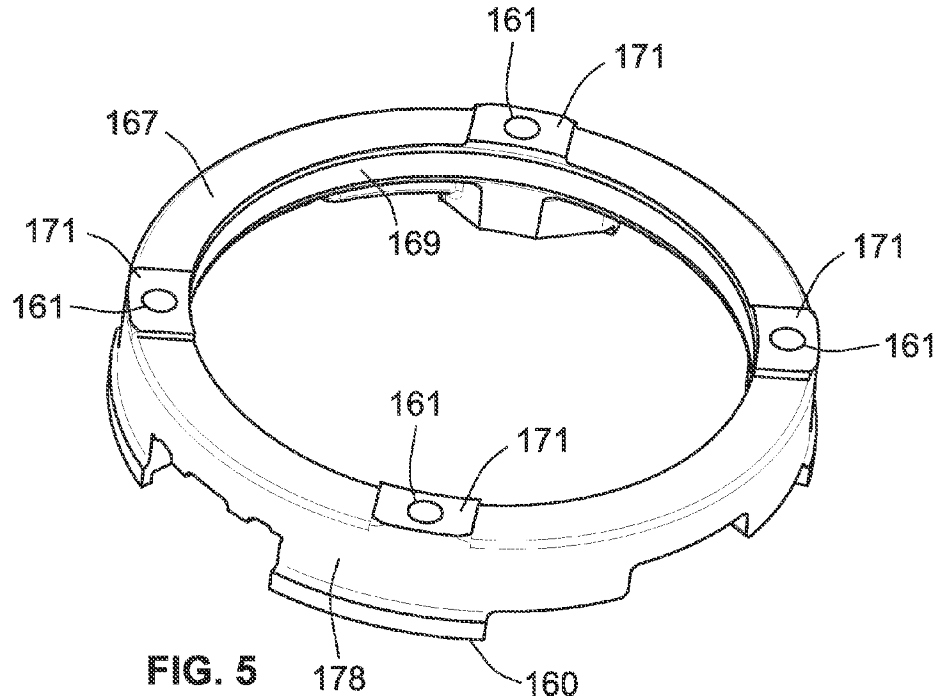
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FIG. 5

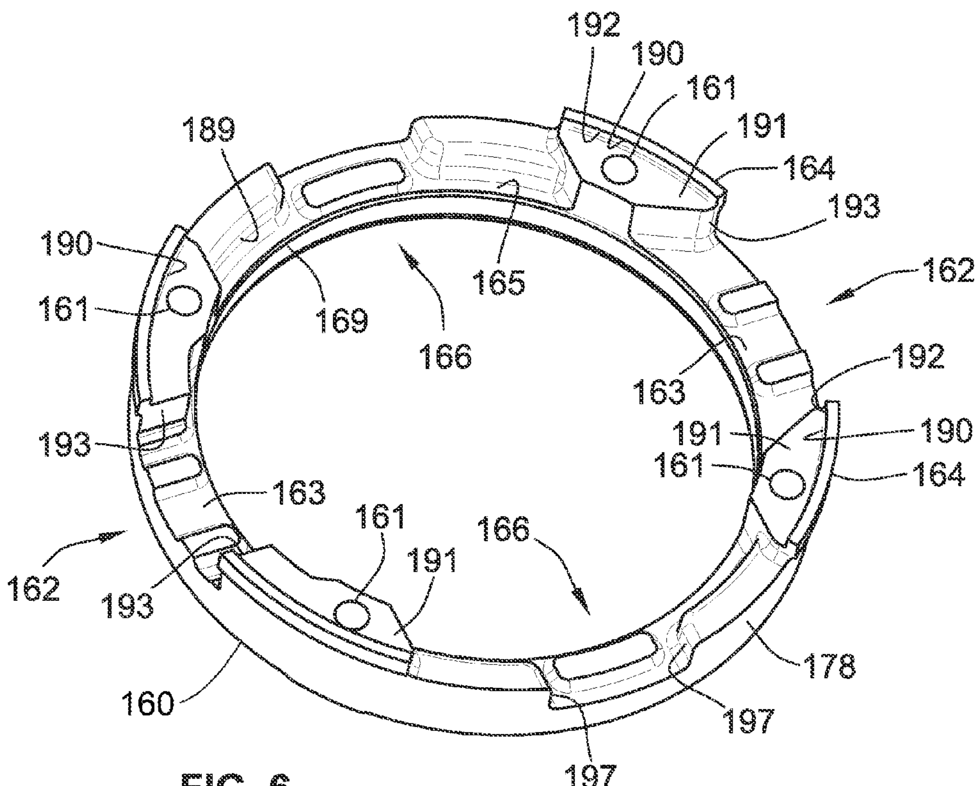
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FIG. 6

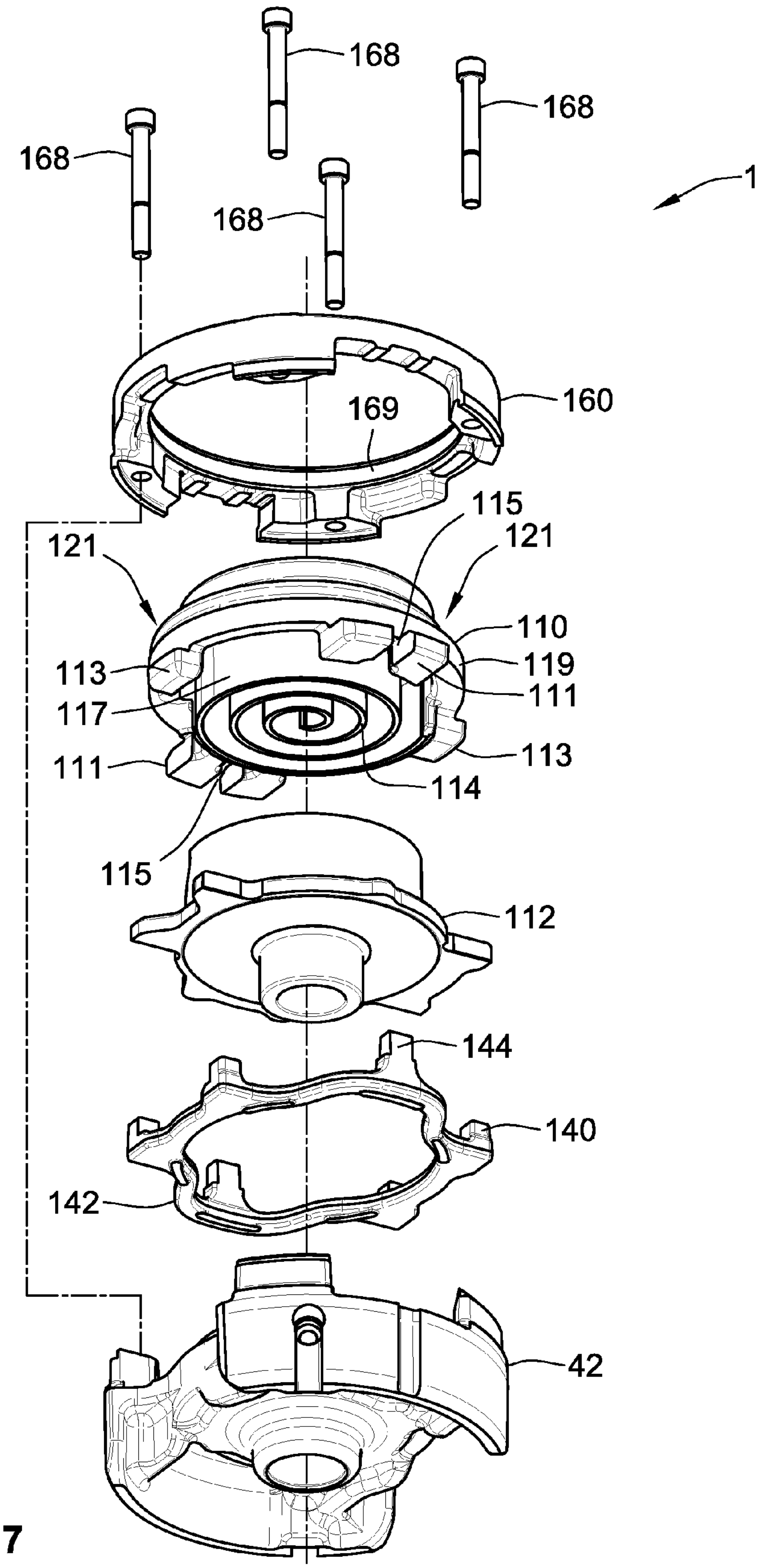
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FIG. 7

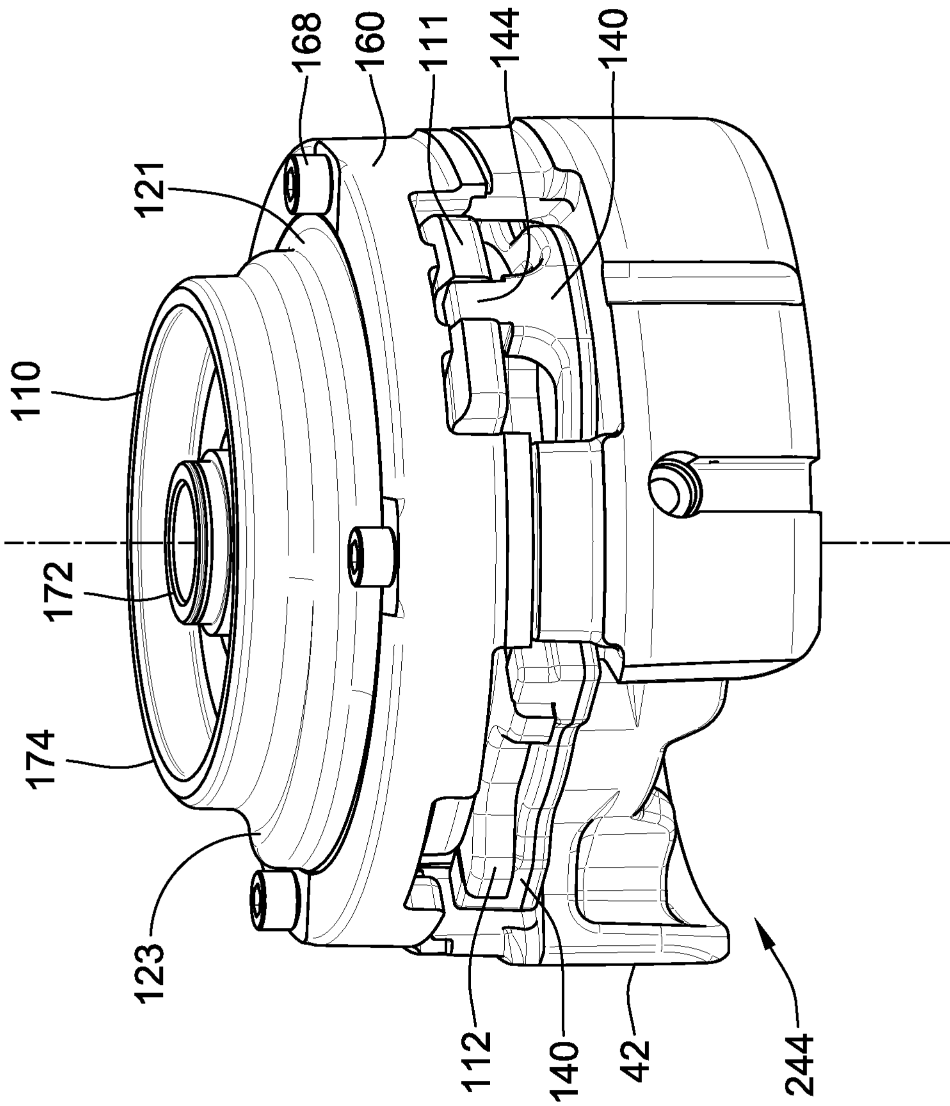


FIG. 8

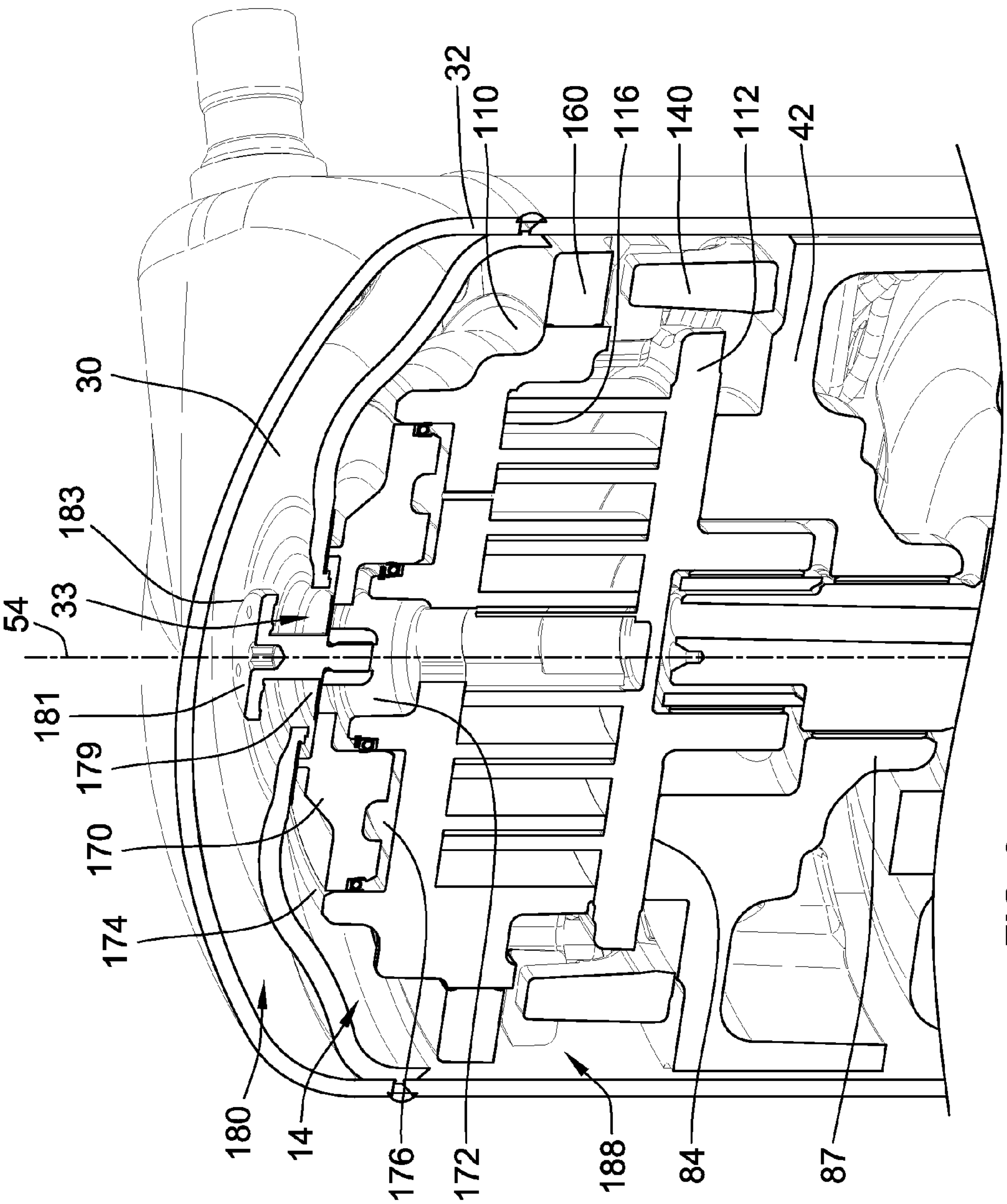
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FIG. 9

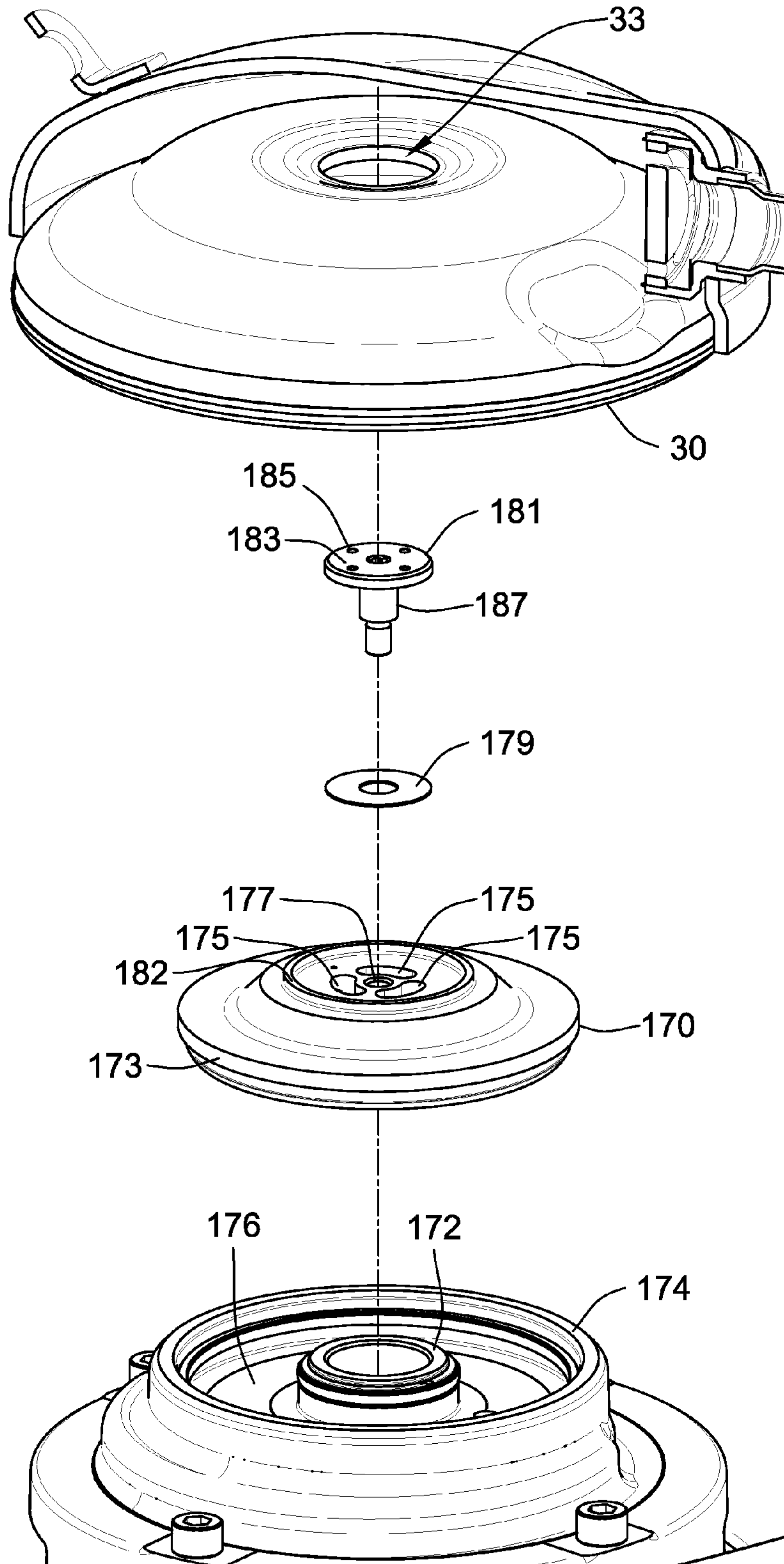


FIG. 10

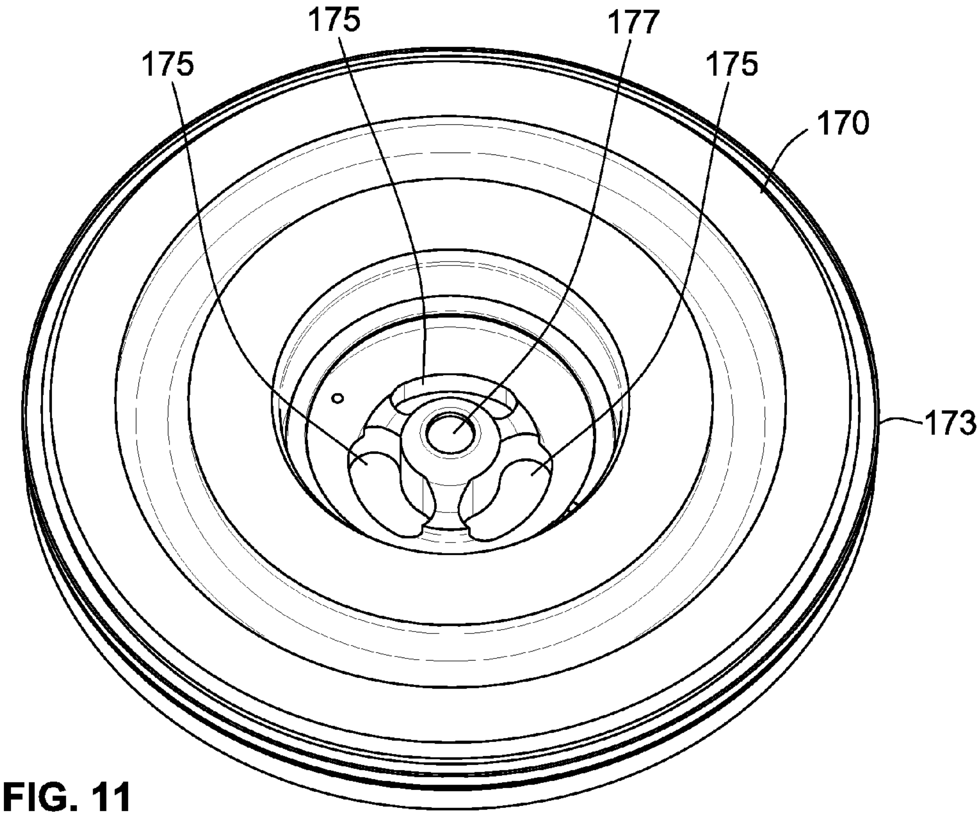


FIG. 11

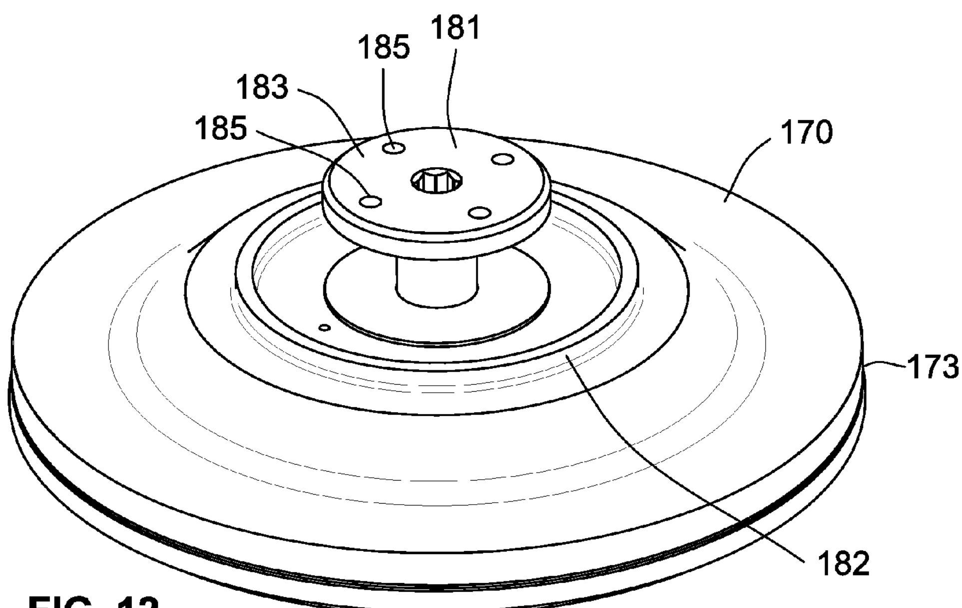


FIG. 12

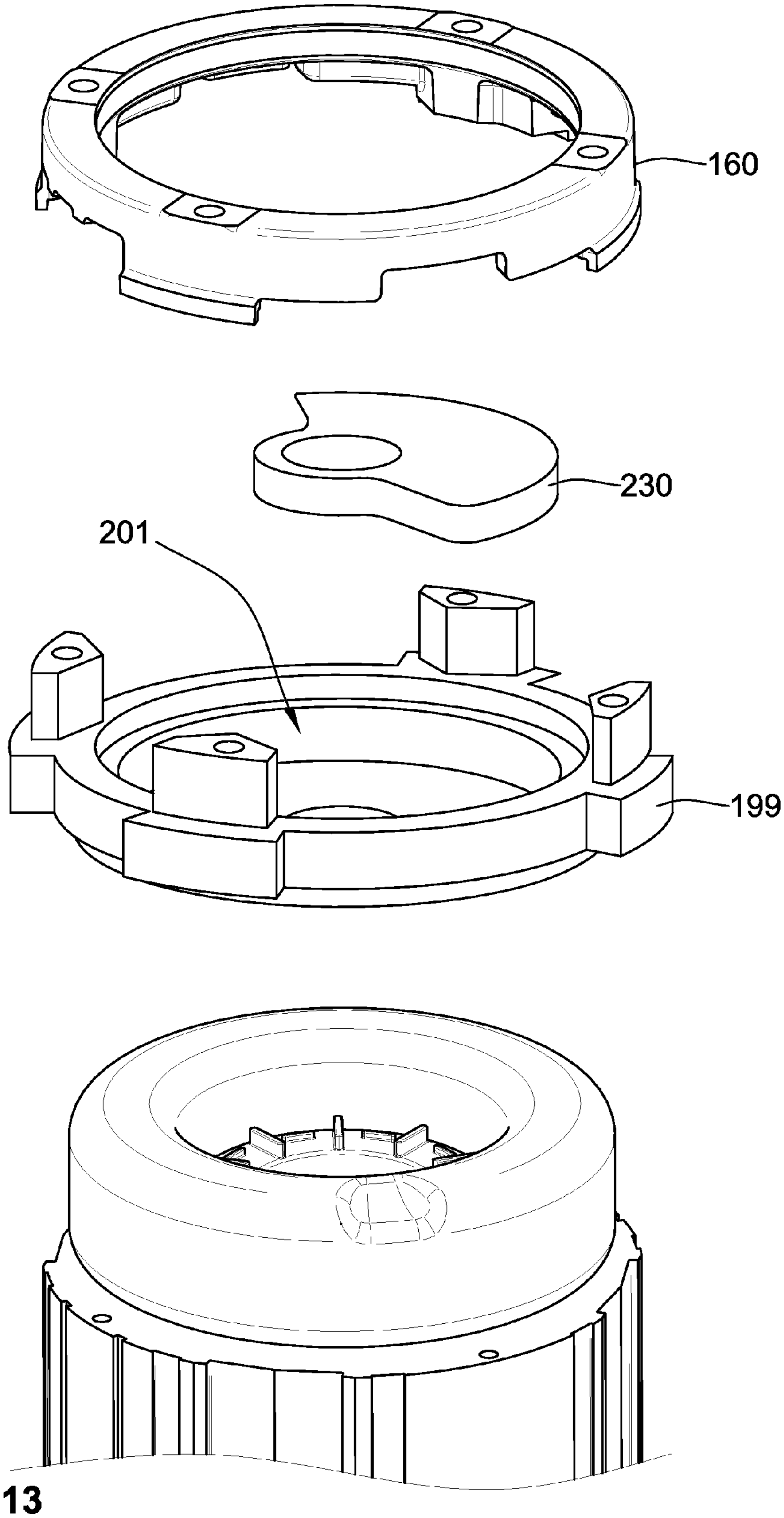


FIG. 13

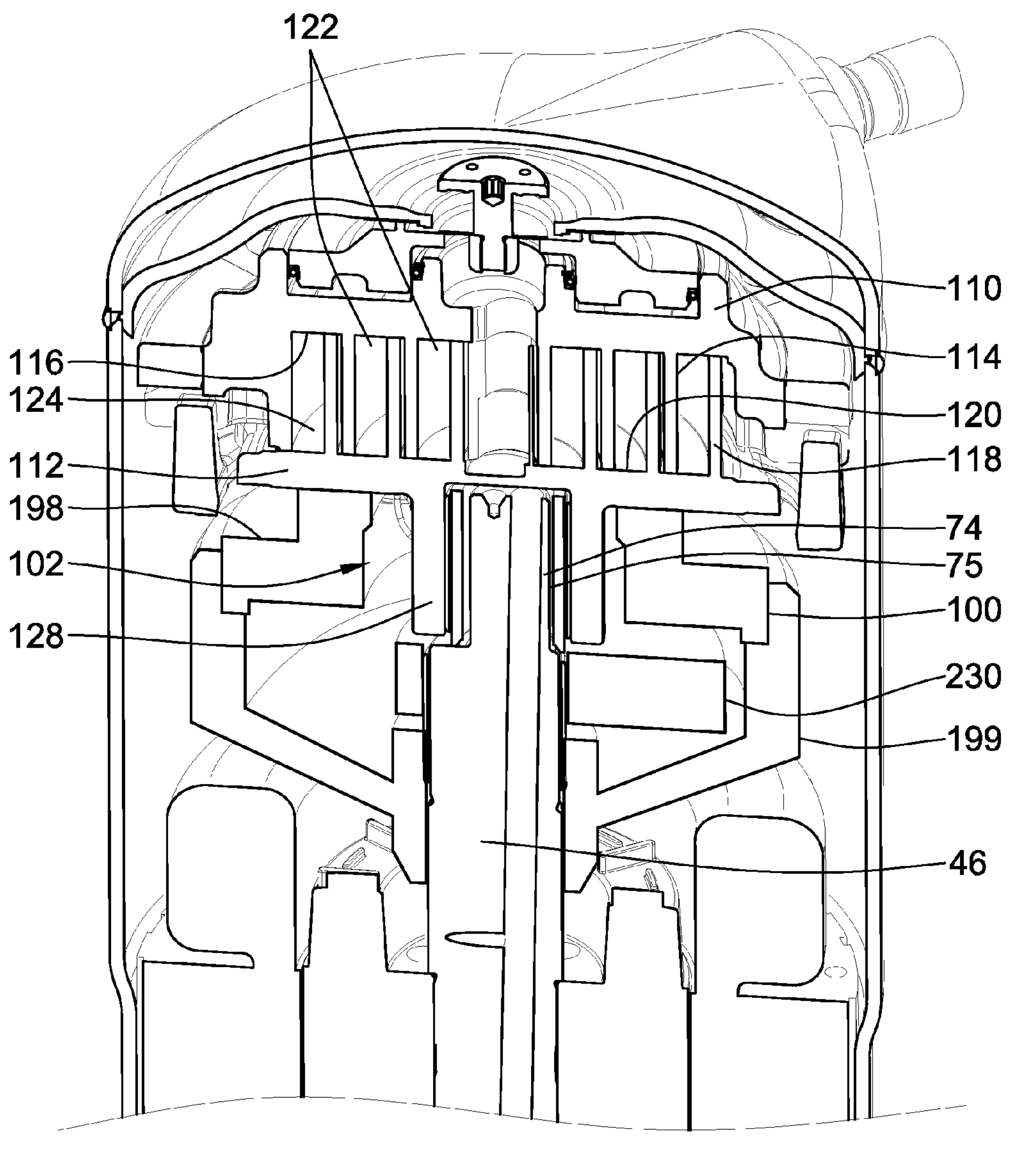


FIG. 14

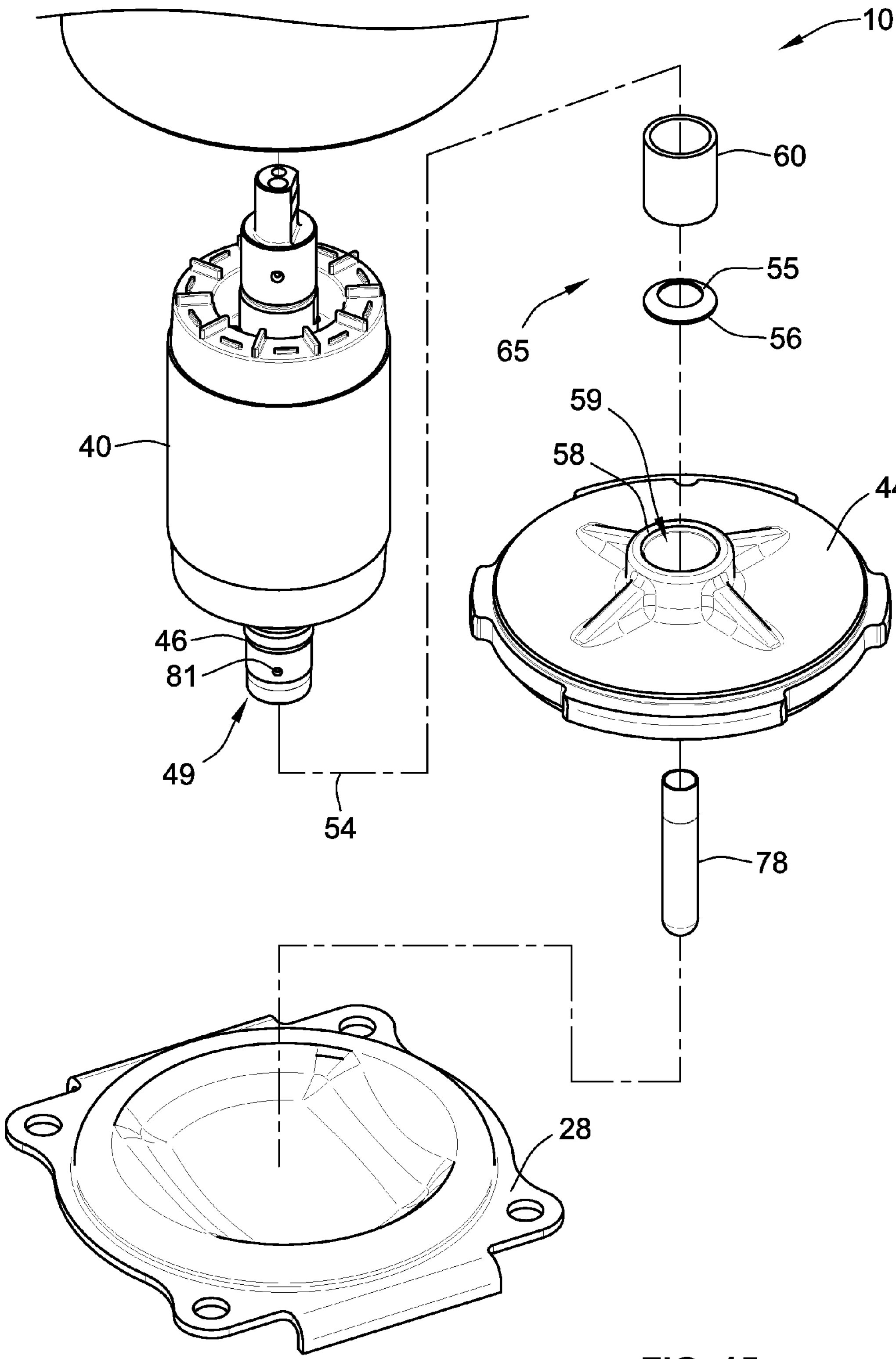


FIG. 15

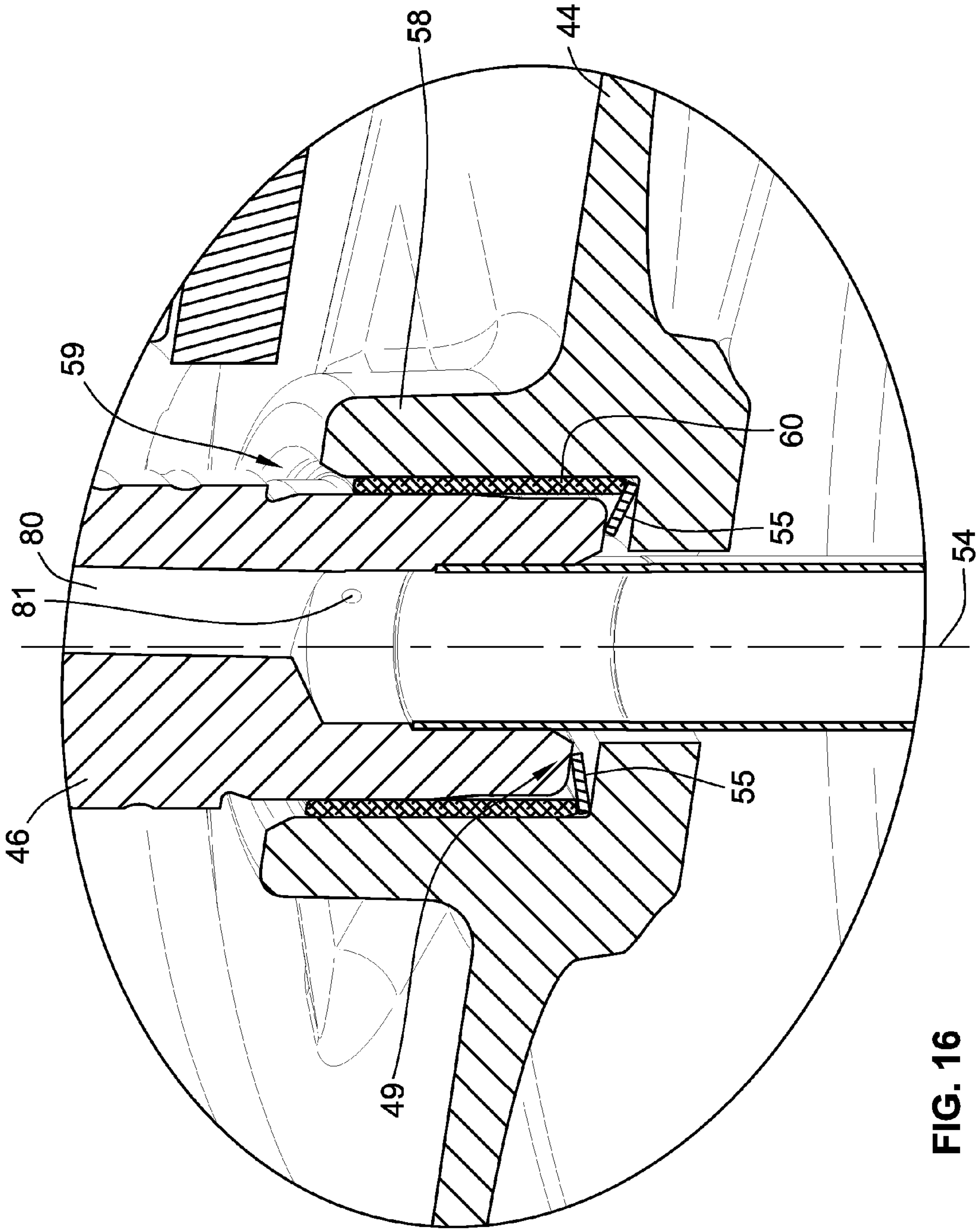


FIG. 16

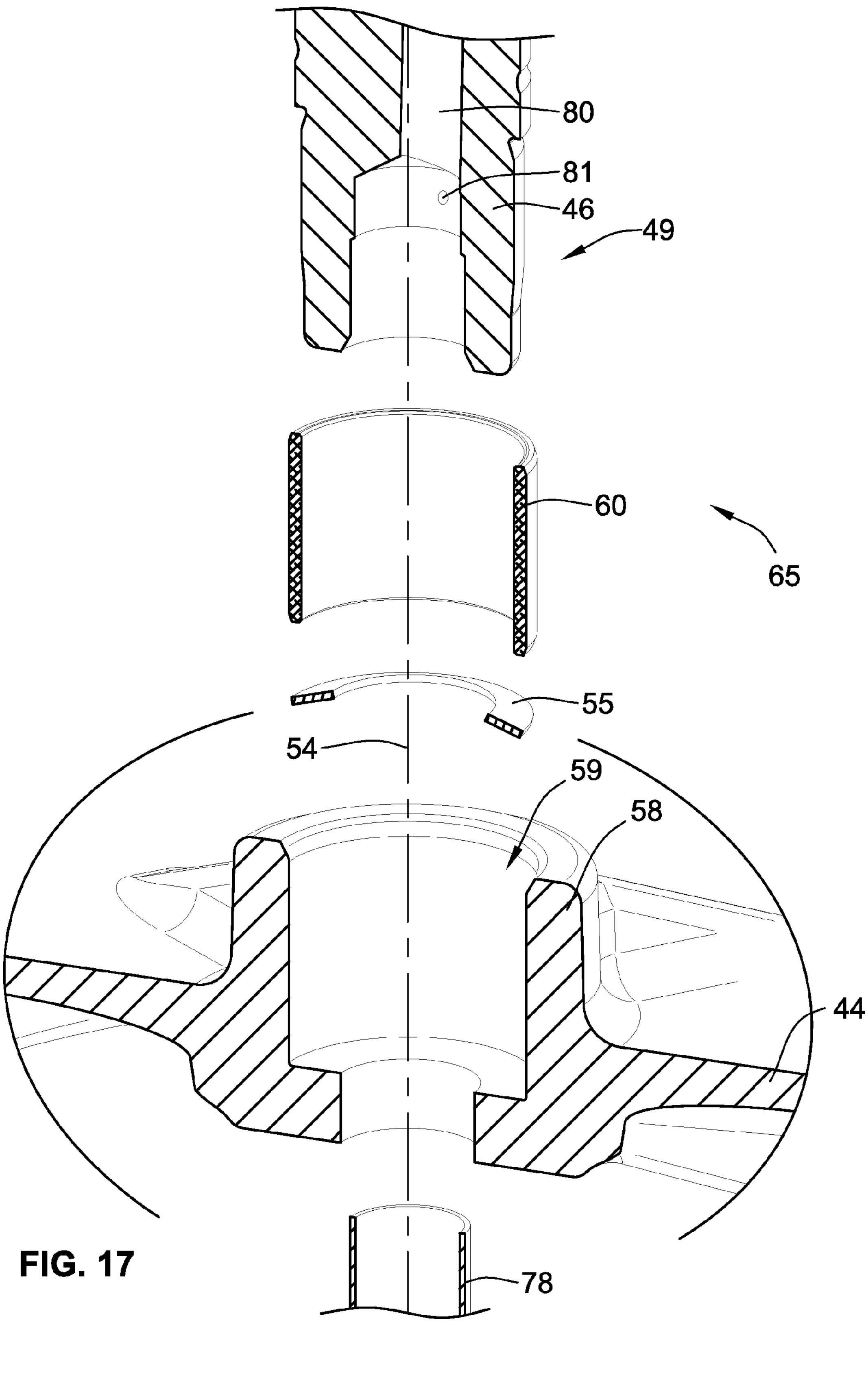


FIG. 17

# SCROLL COMPRESSOR WITH CAPTURED THRUST WASHER

## FIELD OF THE INVENTION

The present invention generally relates to scroll compressors for compressing refrigerant and more particularly to a load transmittal apparatus for transferring an axial load to a thrust surface during operation of the scroll compressor.

## BACKGROUND OF THE INVENTION

A scroll compressor is a certain type of compressor that is used to compress refrigerant for such applications as refrigeration, air conditioning, industrial cooling and freezer applications, and/or other applications where compressed fluid may be used. Such prior scroll compressors are known, for example, as exemplified in U.S. Pat. No. 6,398,530 to Hasemann; U.S. Pat. No. 6,814,551, to Kammhoff et al.; U.S. Pat. No. 6,960,070 to Kammhoff et al.; and U.S. Pat. No. 7,112,046 to Kammhoff et al., all of which are assigned to a Bitzer entity closely related to the present assignee. As the present disclosure pertains to improvements that can be implemented in these or other scroll compressor designs, the entire disclosures of U.S. Pat. Nos. 6,398,530; 7,112,046; 6,814,551; and 6,960,070 are hereby incorporated by reference in their entireties.

As is exemplified by these patents, scroll compressors assemblies conventionally include an outer housing having a scroll compressor contained therein. A scroll compressor includes first and second scroll compressor members. A first compressor member is typically arranged stationary and fixed in the outer housing. A second scroll compressor member is movable relative to the first scroll compressor member in order to compress refrigerant between respective scroll ribs which rise above the respective bases and engage in one another. Conventionally the movable scroll compressor member is driven about an orbital path about a central axis for the purposes of compressing refrigerant. An appropriate drive unit, typically an electric motor, is provided usually within the same housing to drive the movable scroll member.

In some scroll compressors, it is known to have axial restraint, whereby the fixed scroll member has a limited range of movement. This can be desirable due to thermal expansion when the temperature of the orbiting scroll and fixed scroll increases causing these components to expand. Examples of an apparatus to control such restraint are shown in U.S. Pat. No. 5,407,335, issued to Caillat et al., the entire disclosure of which is hereby incorporated by reference.

In a scroll compressor, there is typically some amount of load that is induced in the axial direction of the crankshaft. For a vertical scroll compressor, this load is a combination of the mass of the rotating components as well as any electrically induced load caused by intentional or unintentional axial misalignment of the motor stator and motor rotor. These loads are commonly transmitted between the rotating crankshaft and a stationary housing a thrust surface. The thrust surface may be designed into the stationary component but such surface tends to wear away and surface preparation must be given careful consideration which adds costs to the compressor. It is also known to use a thrust washer, but to prevent unwanted movement, such thrust washer is fixed in place with various ways including the use of fastener(s), adhesive or tabs formed into the circumference of the washer. Such methods add cost to the compressor.

The present disclosure is directed towards improvements over the state of the art as it relates to the above-described features and other features of scroll compressors.

## BRIEF SUMMARY OF THE INVENTION

There is provided a scroll compressor including a load transfer apparatus. The scroll compressor includes a rotating shaft and a stationary lower bearing member. The load transfer apparatus includes a central cylindrical hub defined by the stationary lower bearing member, with the central hub further defining an opening. A cylindrical bearing is configured to seat in the opening. The cylindrical bearing is configured to receive one end of the rotating shaft of the scroll compressor. A thrust washer is disposed in the opening of the central hub and captured axially within the lower bearing member by the cylindrical bearing. An axial load along the center line of the shaft transmits to the stationary lower bearing member through the thrust washer.

A load transfer apparatus of the present disclosure captures the thrust washer in the opening without the use of a fastener or an adhesive. The thrust washer is configured with a smooth circumference, meaning there are no tabs or notches on the circumference of the thrust washer. In one embodiment the thrust washer is metal and in another embodiment the thrust washer is composed of a matrix of a metal, for example steel, bronze, and aluminum, and a polymeric layer, for example PTFE, glass fibers, graphite fibers, silica, molybdenum disulfide or combinations of such material. The cylindrical bearing can also be composed of a metal, and a matrix of metal and a polymeric layer as described above.

There is further provided a scroll compressor including a housing having an upper end and a lower end. A pair of scroll compressor bodies are disposed in the housing. The scroll bodies include a first scroll body and a second scroll body, with the first and second scroll bodies having respective bases and respective scroll ribs that project from the respective bases. The scroll ribs mutually engage each other with the second scroll body being moveable relative to the first scroll body for a compressing fluid.

A pilot ring engages a perimeter surface of the first scroll body to limit movement of the first scroll body in the radial direction. The first scroll body has a first radially-outward-projecting limit tab being configured to limit movement of the first scroll body and at least one of the axial and rotational directions.

A stationary lower bearing member is disposed proximate the lower end of the housing. A motor is disposed in the housing, with the motor including a stator and a rotor with the rotor coupled to a shaft configured to rotate within the housing and with the pair of scroll compressor bodies coupled to the shaft.

A load transfer apparatus includes a central cylindrical hub defined by its stationary lower bearing member with the central hub defining an opening. A cylindrical bearing is configured to seat in the opening, with the cylindrical bearing further configured to receive one end of the shaft. A thrust washer is disposed in the opening of the central hub and captured axially within the lower bearing member by the cylindrical bearing. An axial load along the center line of the shaft is transmitted to the stationary lower bearing member through the thrust washer.

In another embodiment, the pilot ring is formed separately from a crankshaft case, with the pilot ring being attached to a crankcase via a plurality of posts extending axially therebetween. The first and second scroll bodies are disposed within the attached pilot ring and crankcase. A key coupling that acts upon the second scroll body, is disposed within the attached pilot ring and crankcase. The key coupling extends into spaces between adjacent posts, and the spaces allow the pilot ring, crankcase, and key coupling to have outer diameters that are approximately equal to the inner diameter of the housing.

In another aspect, embodiments of the scroll compressor provide a method of transferring axial loading from a rotating shaft in the scroll compressor to a stationary lower bearing member of the scroll compressor. An axial load on the rotating shaft typically includes the mass of the shaft, a motor rotor, and counter weights of the scroll compressor plus electrical-induced loads caused by misalignment of the motor rotor and a motor stator. The method includes depositing a thrust washer at the bottom of an opening in a central cylindrical hub defined by the stationary load bearing member. A cylindrical bearing is inserted into the opening in the central cylindrical hub. The cylindrical bearing is pressed into the opening axially until the bearing captures the thrust washer into position axially in the opening. An end of the shaft is inserted into the cylindrical bearing in the opening defined in the central cylindrical hub, wherein the axial load on the shaft around the center line of the shaft is transmitted to the stationary load bearing member through the thrust washer.

## BRIEF DESCRIPTION OF THE DRAWINGS

The accompanying drawings incorporated in and forming a part of the specification illustrate several aspects of the present invention and, together with the description, serve to explain the principles of the invention. In the drawings:

FIG. **1** is a cross-sectional isometric view of a scroll compressor assembly, according to an embodiment of the invention;

FIG. **2** is a cross-sectional isometric view of an upper portion of the scroll compressor assembly of FIG. **1**;

FIG. **3** is an exploded isometric view of selected components of the scroll compressor assembly of FIG. **1**;

FIG. **4** is a perspective view of an exemplary key coupling and movable scroll compressor body, according to an embodiment of the invention;

FIG. **5** is a top isometric view of the pilot ring, constructed in accordance with an embodiment of the invention;

FIG. **6** is a bottom isometric view of the pilot ring of FIG. **5**;

FIG. **7** is an exploded isometric view of the pilot ring, crankcase, key coupler and scroll compressor bodies, according to an embodiment of the invention;

FIG. **8** is a isometric view of the components of FIG. **7** shown assembled;

FIG. **9** is a cross-sectional isometric view of the components in the top end section of the outer housing, according to an embodiment of the invention;

FIG. **10** is an exploded isometric view of the components of FIG. **9**;

FIG. **11** is a bottom isometric view of the floating seal, according to an embodiment of the invention;

FIG. **12** is a top isometric view of the floating seal of FIG. **11**;

FIG. **13** is an exploded isometric view of selected components for an alternate embodiment of the scroll compressor assembly; and

FIG. **14** is a cross-sectional isometric view of a portion of a scroll compressor assembly, constructed in accordance with an embodiment of the invention.

FIG. **15** is an exploded isometric view of components of the scroll compressor of FIG. **1** including an exemplary embodiment of a load transfer apparatus.

FIG. **16** is a detail assembled cross-section view of the load transfer apparatus illustrated in FIG. **15**.

FIG. **17** is a detail exploded cross-section view of the load transfer apparatus components illustrated in FIG. **15**.

While the invention will be described in connection with certain preferred embodiments, there is no intent to limit it to those embodiments. On the contrary, the intent is to cover all alternatives, modifications and equivalents as included within the spirit and scope of the invention as defined by the appended claims.

## DETAILED DESCRIPTION OF THE INVENTION

An embodiment of the present invention is illustrated in the figures as a scroll compressor assembly **10** generally including an outer housing **12** in which a scroll compressor **14** can be driven by a drive unit **16**. The scroll compressor assembly **10** may be arranged in a refrigerant circuit for refrigeration, industrial cooling, freezing, air conditioning or other appropriate applications where compressed fluid is desired. Appropriate connection ports provide for connection to a refrigeration circuit and include a refrigerant inlet port **18** and a refrigerant outlet port **20** extending through the outer housing **12**. The scroll compressor assembly **10** is operable through operation of the drive unit **16** to operate the scroll compressor **14** and thereby compress an appropriate refrigerant or other fluid that enters the refrigerant inlet port **18** and exits the refrigerant outlet port **20** in a compressed high-pressure state.

The outer housing for the scroll compressor assembly **10** may take many forms. In particular embodiments of the invention, the outer housing **12** includes multiple shell sections. In the embodiment of FIG. **1**, the outer housing **12** includes a central cylindrical housing section **24**, and a top end housing section **26**, and a single-piece bottom shell **28** that serves as a mounting base. In certain embodiments, the housing sections **24**, **26**, **28** are formed of appropriate sheet steel and welded together to make a permanent outer housing **12** enclosure. However, if disassembly of the housing is desired, other housing assembly provisions can be made that can include metal castings or machined components, wherein the housing sections **24**, **26**, **28** are attached using fasteners.

As can be seen in the embodiment of FIG. **1**, the central housing section **24** is cylindrical, joined with the top end housing section **26**. In this embodiment, a separator plate **30** is disposed in the top end housing section **26**. During assembly, these components can be assembled such that when the top end housing section **26** is joined to the central cylindrical housing section **24**, a single weld around the circumference of the outer housing **12** joins the top end housing section **26**, the separator plate **30**, and the central cylindrical housing section **24**. In particular embodiments, the central cylindrical housing section **24** is welded to the single-piece bottom shell **28**, though, as stated above, alternate embodiments would include other methods of joining (e.g., fasteners) these sections of the outer housing **12**. Assembly of the outer housing **12** results in the formation of an enclosed chamber **31** that surrounds the drive unit **16**, and partially surrounds the scroll compressor **14**. In particular embodiments, the top end housing section **26** is generally dome-shaped and includes a respective cylindrical side wall region **32** that abuts the top of the central cylindrical housing section **24**, and provides for closing off the top end of the outer housing **12**. As can also be seen from FIG. **1**, the bottom of the central cylindrical housing section **24** abuts a flat portion just to the outside of a raised annular rib **34** of the bottom end housing section **28**. In at least one embodiment of the invention, the central cylindrical housing section **24** and bottom end housing section **28** are joined by an exterior weld around the circumference of a bottom end of the outer housing **12**.

In a particular embodiment, the drive unit **16** in is the form of an electrical motor assembly **40**. The electrical motor assembly **40** operably rotates and drives a shaft **46**. Further, the electrical motor assembly **40** generally includes a stator **50** comprising electrical coils and a rotor **52** that is coupled to the drive shaft **46** for rotation together. The stator **50** is supported by the outer housing **12**, either directly or via an adapter. The stator **50** may be press-fit directly into outer housing **12**, or may be fitted with an adapter (not shown) and press-fit into the outer housing **12**. In a particular embodiment, the rotor **52** is mounted on the drive shaft **46**, which is supported by upper and lower bearings **42**, **44**. Energizing the stator **50** is operative to rotatably drive the rotor **52** and thereby rotate the drive shaft **46** about a central axis **54**. Applicant notes that when the terms “axial” and “radial” are used herein to describe features of components or assemblies, they are defined with respect to the central axis **54**. Specifically, the term “axial” or “axially-extending” refers to a feature that projects or extends in a direction parallel to the central axis **54**, while the terms “radial’ or “radially-extending” indicates a feature that projects or extends in a direction perpendicular to the central axis **54**.

In one embodiment an axial load induced along the centerline **54** of the crankshaft **46** is transferred to the stationary lower bearing member **44** by a load transfer apparatus **65**.

Referring to FIGS. **15-17**, an exemplary embodiment of a load transfer apparatus **65** is illustrated in an assembled view and an exploded view. A central cylindrical hub **58** is defined in the lower bearing member **44**, with the cylindrical hub **58** further defining an opening **59**. The opening is configured to receive one end **49** of the shaft **46** and a cylindrical bearing **60**. The bearing **60** is lubricated by oil through an orifice **81** defined in the shaft **46**. The orifice **81** is in fluid communication with the internal lubricant passageway **80** defined by the shaft **46**.

A thrust washer **55** is disposed in the opening **59** at the bottom of the central cylindrical hub **58** (See FIG. **16**). The thrust washer **55** is disposed between the cylindrical bearing **60** and the stationary lower bearing **44**. In one configuration the thrust washer **55** is captured in the opening **59** by the cylindrical bearing **60**. During compressor **14** operation, since the friction between the shaft **46** and thrust washer **55** is substantially less than the friction between the thrust washer **55** at the bearing housing **44** the thrust washer **55** will remain stationary, i.e. will not spin with the shaft **46** or move axially. In another configuration the cylindrical bearing **60** is pressed into the opening **59** axially until sufficient force is exerted against the thrust washer **55** to capture the thrust washer in position axially but allow the thrust washer **55** to rotate since there is some axial clearance between the cylindrical bearing and the washer. With the load transfer apparatus **65**, there is no need to fix the thrust washer **55** in position with adhesive, fasteners or other means, for example tabs defined on the circumference of the thrust washer **55**. The thrust washer **55** in the described load transfer apparatus **65** is configured with a smooth circumference, i.e. no tabs, grooves or projections. The thrust washer **55** is composed of one of a metal or a metal and a polymeric layer capable of transferring the axial load from the shaft **46** to the lower bearing **44**.

The two bearings, cylindrical **60** or thrust washer **55**, can be either all metal or a metal-nonmetal assemblage. In a typical configuration, either or both bearings are composed of three layers. The outermost (away from the load bearing surface) is steel (to provide structural strength. To this is bonded a layer of sintered bronze particles in a “loose” (i.e. porous) matrix. Finally a polymeric layer is bonded into the porous matrix. The polymeric layer may also include PTFE, glass fibers or particles, graphite fibers or particles, silica, molybdenum disulfide, and/or other fillers. Alternately, all-metal bearings will typically have the steel shell and a solid bronze or babbitt liner. Some others may have a steel shell and porous bronze liner with a polymer or PTFE filing the bronze matrix but not forming an actual layer on top of the bronze. Another configuration is a bearing made of a single metal, without the described layered construction. In this case the material is typically a bronze or aluminum alloy.

The axial load is typically the combination of the mass of the rotating components that include the shaft **46**, the motor rotor **52** and counter weight and other members coupled to the shaft **46**. The axial load also includes any electrical induced load caused by intentional or unintentional axial misalignment of the motor stator **50** and motor rotor **52**.

With reference to FIG. **1**, the lower bearing member **44** includes a central, generally cylindrical hub **58** that includes a central bushing **58** and opening **59** to provide the cylindrical bearing **60** to which the drive shaft **46** is journaled for rotational support. A plate-like ledge region **68** of the lower bearing member **44** projects radially outward from the central hub **58**, and serves to separate a lower portion of the stator **50** from an oil lubricant sump **76**. An axially-extending perimeter surface **70** of the lower bearing member **44** may engage with the inner diameter surface of the central housing section **24** to centrally locate the lower bearing member **44** and thereby maintain its position relative to the central axis **54**. This can be by way of an interference and press-fit support arrangement between the lower bearing member **44** and the outer housing **12**.

In the embodiment of FIG. **1**, the drive shaft **46** has an impeller tube **47** attached at the bottom end of the drive shaft **46**. In a particular embodiment, the impeller tube **47** is of a smaller diameter than the drive shaft **46**, and is aligned concentrically with the central axis **54**. As can be seen from FIG. **1**, the drive shaft **46** and impeller tube **47** pass through an opening in the cylindrical hub **58** of the lower bearing member **44**. At its upper end, the drive shaft **46** is journaled for rotation within the upper bearing member **42**. Upper bearing member **42** may also be referred to as a “crankcase.”

The drive shaft **46** further includes an offset eccentric drive section **74** that has a cylindrical drive surface **75** (shown in FIG. **2**) about an offset axis that is offset relative to the central axis **54**. This offset drive section **74** is journaled within a cavity of a movable scroll compressor body **112** of the scroll compressor **14** to drive the movable scroll compressor body **112** about an orbital path when the drive shaft **46** rotates about the central axis **54**. To provide for lubrication of all of the various bearing surfaces, the outer housing **12** provides the oil lubricant sump **76** at the bottom end of the outer housing **12** in which suitable oil lubricant is provided. The impeller tube **47** has an oil lubricant passage and inlet port **78** formed at the end of the impeller tube **47**. Together, the impeller tube **47** and inlet port **78** act as an oil pump when the drive shaft **46** is rotated, and thereby pumps oil out of the lubricant sump **76** into an internal lubricant passageway **80** defined within the drive shaft **46**. During rotation of the drive shaft **46**, centrifugal force acts to drive lubricant oil up through the lubricant passageway **80** against the action of gravity. The lubricant passageway **80** has various radial passages projecting therefrom to feed oil through centrifugal force to appropriate bearing surfaces and thereby lubricate sliding surfaces as may be desired.

As shown in FIGS. **2** and **3**, the upper bearing member, or crankcase, **42** includes a central bearing hub **87** into which the drive shaft **46** is journaled for rotation, and a thrust bearing **84** that supports the movable scroll compressor body **112**. (See also FIG. **9**). Extending outward from the central bearing hub **87** is a disk-like portion **86** that terminates in an intermittent perimeter support surface **88** defined by discretely spaced posts **89**. In the embodiment of FIG. **3**, the central bearing hub **87** extends below the disk-like portion **86**, while the thrust bearing **84** extends above the disk-like portion **86**. In certain embodiments, the intermittent perimeter support surface **88** is adapted to have an interference and press-fit with the outer housing **12**. In the embodiment of FIG. **3**, the crankcase **42** includes four posts **89**, each post having an opening **91** configured to receive a threaded fastener. It is understood that alternate embodiments of the invention may include a crankcase with more or less than four posts, or the posts may be separate components altogether. Alternate embodiments of the invention also include those in which the posts are integral with the pilot ring instead of the crankcase.

In certain embodiments such as the one shown in FIG. **3**, each post **89** has an arcuate outer surface **93** spaced radially inward from the inner surface of the outer housing **12**, angled interior surfaces **95**, and a generally flat top surface **97** which can support a pilot ring **160**. In this embodiment, intermittent perimeter support surface **88** abuts the inner surface of the outer housing **12**. Further, each post **89** has a chamfered edge **94** on a top, outer portion of the post **89**. In particular embodiments, the crankcase **42** includes a plurality of spaces **244** between adjacent posts **89**. In the embodiment shown, these spaces **244** are generally concave and the portion of the crankcase **42** bounded by these spaces **244** will not contact the inner surface of the outer housing **12**.

The upper bearing member or crankcase **42** also provides axial thrust support to the movable scroll compressor body **112** through a bearing support via an axial thrust surface **96** of the thrust bearing **84**. While, as shown FIGS. **1-3**, the crankcase **42** may be integrally provided by a single unitary component, FIGS. **13** and **14** show an alternate embodiment in which the axial thrust support is provided by a separate collar member **198** that is assembled and concentrically located within the upper portion of the upper bearing member **199** along stepped annular interface **100**. The collar member **198** defines a central opening **102** that is a size large enough to clear a cylindrical bushing drive hub **128** of the movable scroll compressor body **112** in addition to the eccentric offset drive section **74**, and allow for orbital eccentric movement thereof.

Turning in greater detail to the scroll compressor **14**, the scroll compressor includes first and second scroll compressor bodies which preferably include a stationary fixed scroll compressor body **110** and a movable scroll compressor body **112**. While the term "fixed" generally means stationary or immovable in the context of this application, more specifically "fixed" refers to the non-orbiting, non-driven scroll member, as it is acknowledged that some limited range of axial, radial, and rotational movement is possible due to thermal expansion and/or design tolerances.

The movable scroll compressor body **112** is arranged for orbital movement relative to the fixed scroll compressor body **110** for the purpose of compressing refrigerant. The fixed scroll compressor body includes a first rib **114** projecting axially from a plate-like base **116** and is designed in the form of a spiral. Similarly, the movable scroll compressor body **112** includes a second scroll rib **118** projecting axially from a plate-like base **120** and is in the shape of a similar spiral. The scroll ribs **114**, **118** engage in one another and abut sealingly on the respective surfaces of bases **120**, **116** of the respectively other compressor body **112**, **110**. As a result, multiple compression chambers **122** are formed between the scroll ribs **114**, **118** and the bases **120**, **116** of the compressor bodies **112**, **110**. Within the chambers **122**, progressive compression of refrigerant takes place. Refrigerant flows with an initial low pressure via an intake area **124** surrounding the scroll ribs **114**, **118** in the outer radial region (see e.g. FIGS. **1-2**). Following the progressive compression in the chambers **122** (as the chambers progressively are defined radially inward), the refrigerant exits via a compression outlet **126** which is defined centrally within the base **116** of the fixed scroll compressor body **110**. Refrigerant that has been compressed to a high pressure can exit the chambers **122** via the compression outlet **126** during operation of the scroll compressor **14**.

The movable scroll compressor body **112** engages the eccentric offset drive section **74** of the drive shaft **46**. More specifically, the receiving portion of the movable scroll compressor body **112** includes the cylindrical bushing drive hub **128** which slideably receives the eccentric offset drive section **74** with a slideable bearing surface provided therein. In detail, the eccentric offset drive section **74** engages the cylindrical bushing drive hub **128** in order to move the movable scroll compressor body **112** about an orbital path about the central axis **54** during rotation of the drive shaft **46** about the central axis **54**. Considering that this offset relationship causes a weight imbalance relative to the central axis **54**, the assembly typically includes a counterweight **130** that is mounted at a fixed angular orientation to the drive shaft **46**. The counterweight **130** acts to offset the weight imbalance caused by the eccentric offset drive section **74** and the movable scroll compressor body **112** that is driven about an orbital path. The counterweight **130** includes an attachment collar **132** and an offset weight region **134** (see counterweight **130** shown best in FIGS. **2** and **3**) that provides for the counterweight effect and thereby balancing of the overall weight of the components rotating about the central axis **54**. This provides for reduced vibration and noise of the overall assembly by internally balancing or cancelling out inertial forces.

With reference to FIGS. **4** and **7**, the guiding movement of the scroll compressor **14** can be seen. To guide the orbital movement of the movable scroll compressor body **112** relative to the fixed scroll compressor body **110**, an appropriate key coupling **140** may be provided. Keyed couplings **140** are often referred to in the scroll compressor art as an "Oldham Coupling." In this embodiment, the key coupling **140** includes an outer ring body **142** and includes two axially-projecting first keys **144** that are linearly spaced along a first lateral axis **146** and that slide closely and linearly within two respective keyway tracks or slots **115** (shown in FIGS. **1** and **2**) of the fixed scroll compressor body **110** that are linearly spaced and aligned along the first axis **146** as well. The slots **115** are defined by the stationary fixed scroll compressor body **110** such that the linear movement of the key coupling **140** along the first lateral axis **146** is a linear movement relative to the outer housing **12** and perpendicular to the central axis **54**. The keys can comprise slots, grooves or, as shown, projections which project axially (i.e., parallel to central axis **54**) from the ring body **142** of the key coupling **140**. This control of movement along the first lateral axis **146** guides part of the overall orbital path of the movable scroll compressor body **112**.

Referring specifically to FIG. **4**, the key coupling **140** includes four axially-projecting second keys **152** in which opposed pairs of the second keys **152** are linearly aligned substantially parallel relative to a second transverse lateral axis **154** that is perpendicular to the first lateral axis **146**. There are two sets of the second keys **152** that act cooperatively to receive projecting sliding guide portions **254** that project from the base **120** on opposite sides of the movable scroll compressor body **112**. The guide portions **254** linearly engage and are guided for linear movement along the second transverse lateral axis by virtue of sliding linear guiding movement of the guide portions **254** along sets of the second keys **152**.

It can be seen in FIG. **4** that four sliding contact surfaces **258** are provided on the four axially-projecting second keys **152** of the key coupling **140**. As shown, each of the sliding contact surfaces **258** is contained in its own separate quadrant **252** (the quadrants **252** being defined by the mutually perpendicular lateral axes **146**, **154**). As shown, cooperating pairs of the sliding contact surfaces **258** are provided on each side of the first lateral axis **146**.

By virtue of the key coupling **140**, the movable scroll compressor body **112** has movement restrained relative to the fixed scroll compressor body **110** along the first lateral axis **146** and second transverse lateral axis **154**. This results in the prevention of relative rotation of the movable scroll body as it allows only translational motion. More particularly, the fixed scroll compressor body **110** limits motion of the key coupling **140** to linear movement along the first lateral axis **146**; and in turn, the key coupling **140** when moving along the first lateral axis **146** carries the movable scroll **112** along the first lateral axis **146** therewith. Additionally, the movable scroll compressor body can independently move relative to the key coupling **140** along the second transverse lateral axis **154** by virtue of relative sliding movement afforded by the guide portions **254** which are received and slide between the second keys **152**. By allowing for simultaneous movement in two mutually perpendicular axes **146**, **154**, the eccentric motion that is afforded by the eccentric offset drive section **74** of the drive shaft **46** upon the cylindrical bushing drive hub **128** of the movable scroll compressor body **112** is translated into an orbital path movement of the movable scroll compressor body **112** relative to the fixed scroll compressor body **110**.

To carry axial thrust loads, the movable scroll compressor body **112** also includes flange portions **268** projecting in a direction perpendicular relative to the guiding flange portions **262** (e.g. along the first lateral axis **146**). These additional flange portions **268** are preferably contained within the diametrical boundary created by the guide flange portions **262** so as to best realize the size reduction benefits. Yet a further advantage of this design is that the sliding faces **254** of the movable scroll compressor body **112** are open and not contained within a slot. This is advantageous during manufacture in that it affords subsequent machining operations such as finishing milling for creating the desirable tolerances and running clearances as may be desired.

Generally, scroll compressors with movable and fixed scroll compressor bodies require some type of restraint for the fixed scroll compressor body **110** which restricts the radial movement and rotational movement but which allows some degree of axial movement so that the fixed and movable scroll compressor bodies **110**, **112** are not damaged during operation of the scroll compressor **14**. In embodiments of the invention, that restraint is provided by a pilot ring **160**, as shown in FIGS. **5-9**. FIG. **5** shows the top side of pilot ring **160**, constructed in accordance with an embodiment of the invention. The pilot ring **160** has a top surface **167**, a cylindrical outer perimeter surface **178**, and a cylindrical first inner wall **169**. The pilot ring **160** of FIG. **5** includes four holes **161** through which fasteners, such as threaded bolts, may be inserted to allow for attachment of the pilot ring **160** to the crankcase **42**. In a particular embodiment, the pilot ring **160** has axially-raised portions **171** (also referred to as mounting bosses) where the holes **161** are located. One of skill in the art will recognize that alternate embodiments of the pilot ring may have greater or fewer than four holes for fasteners. The pilot ring **160** may be a machined metal casting, or, in alternate embodiments, a machined component of iron, steel, aluminum, or some other similarly suitable material.



FIG. **6** shows a bottom view of the pilot ring **160** showing the four holes **161** along with two slots **162** formed into the pilot ring **160**. In the embodiment of FIG. **6**, the slots **162** are spaced approximately 180 degrees apart on the pilot ring **160**. Each slot **162** is bounded on two sides by axially-extending side walls **193**. As shown in FIG. **6**, the bottom side of the pilot ring **160** includes a base portion **163** which is continuous around the entire circumference of the pilot ring **160** forming a complete cylinder. But on each side of the two slots **162**, there is a semi-circular stepped portion **164** which covers some of the base portion **163** such that a ledge **165** is formed on the part of the pilot ring **160** radially inward of each semi-circular stepped portion **164**. The inner-most diameter or the ledge **165** is bounded by the first inner wall **169**.

A second inner wall **189** runs along the inner diameter of each semi-circular stepped portion **164**. Each semi-circular stepped portion **164** further includes a bottom surface **191**, a notched section **166**, and a chamfered lip **190**. In the embodiment of FIG. **6**, each chamfered lip **190** runs the entire length of the semi-circular stepped portion **164** making the chamfered lip **190** semi-circular as well. Each chamfered lip **190** is located on the radially-outermost edge of the bottom surface **191**, and extends axially from the bottom surface **191**. Further, each chamfered lip **190** includes a chamfered edge surface **192** on an inner radius of the chamfered lip **190**. When assembled, the chamfered edge surface **192** is configured to mate with the chamfered edge **94** on each post **89** of the crankcase. The mating of these chamfered surfaces allows for an easier, better-fitting assembly, and reduces the likelihood of assembly problems due to manufacturing tolerances.

In the embodiment of FIG. **6**, the notched sections **166** are approximately 180 degrees apart on the pilot ring **160**, and each is about midway between the two ends of the semi-circular stepped portion **164**. The notched sections **166** are bounded on the sides by sidewall sections **197**. Notched sections **166** thus extend radially and axially into the semi-circular stepped portion **164** of the pilot ring **160**.

FIG. **7** shows an exploded view of the scroll compressor **14** assembly, according to an embodiment of the invention. The top-most component shown is the pilot ring **160** which is adapted to fit over the top of the fixed scroll compressor body **110**. The fixed scroll compressor body **110** has a pair of first radially-outward projecting limit tabs **111**. In the embodiment of FIG. **7**, one of the pair of first radially-outward projecting limit tabs **111** is attached to an outermost perimeter surface **117** of the first scroll rib **114**, while the other of the pair of first radially-outward projecting limit tabs **111** is attached to a perimeter portion of the fixed scroll compressor body **110** below a perimeter surface **119**. In further embodiments, the pair of first radially-outward projecting limit tabs **111** are spaced approximately 180 degrees apart. Additionally, in particular embodiments, each of the pair of first radially-outward-projecting limit tabs **111** has a slot **115** therein. In particular embodiments, the slot **115** may be a U-shaped opening, a rectangular-shaped opening, or have some other suitable shape.

The fixed scroll compressor body **110** also has a pair of second radially-outward projecting limit tabs **113**, which, in this embodiment, are spaced approximately 180 degrees apart. In certain embodiments, the second radially-outward projecting limit tabs **113** share a common plane with the first radially-outward-projecting limit tabs **111**. Additionally, in the embodiment of FIG. **7**, one of the pair of second radially-outward projecting limit tabs **113** is attached to an outermost perimeter surface **117** of the first scroll rib **114**, while the other of the pair of second radially-outward projecting limit tabs **113** is attached to a perimeter portion of the fixed scroll compressor body **110** below the perimeter surface **119**. The movable scroll compressor body **112** is configured to be held within the keys of the key coupling **140** and mates with the fixed scroll compressor body **110**. As explained above, the key coupling **140** has two axially-projecting first keys **144**, which are configured to be received within the slots **115** in the first radially-outward-projecting limit tabs **111**. When assembled, the key coupling **140**, fixed and movable scroll compressor bodies **110**, **112** are all configured to be disposed within crankcase **42**, which can be attached the to the pilot ring **160** by the threaded bolts **168** shown above the pilot ring **160**.

Referring still to FIG. **7**, the fixed scroll compressor body **110** includes plate-like base **116** (see FIG. **14**) and a perimeter surface **119** spaced axially from the plate-like base **116**. In a particular embodiment, the entirety of the perimeter surface **119** surrounds the first scroll rib **114** of the fixed scroll compressor body **110**, and is configured to abut the first inner wall **169** of the pilot ring **160**, though embodiments are contemplated in which the engagement of the pilot ring and fixed scroll compressor body involve less than the entire circumference. In particular embodiments of the invention, the first inner wall **169** is precisely toleranced to fit snugly around the perimeter surface **119** to thereby limit radial movement of the first scroll compressor body **110**, and thus provide radial restraint for the first scroll compressor body **110**. The plate-like base **116** further includes a radially-extending top surface **121** that extends radially inward from the perimeter surface **119**. The radially-extending top surface **121** extends radially inward towards a step-shaped portion **123** (see FIG. **8**). From this step-shaped portion **123**, a cylindrical inner hub region **172** and peripheral rim **174** extend axially (i.e., parallel to central axis **54**, when assembled into scroll compressor assembly **10**).

FIG. **8** shows the components of FIG. **7** fully assembled. The pilot ring **160** securely holds the fixed scroll compressor body **110** in place with respect to the movable scroll compressor body **112** and key coupling **140**. The threaded bolts **168** attach the pilot ring **160** and crankcase **42**. As can be seen from FIG. **8**, each of the pair of first radially-outward projecting limit tabs **111** is positioned in its respective slot **162** of the pilot ring **160**. As stated above, the slots **115** in the pair of first radially-outward projecting limit tabs **111** are configured to receive the two axially-projecting first keys **144**. In this manner, the pair of first radially-outward projecting limit tabs **111** engage the side portion **193** of the pilot ring slots **162** to prevent rotation of the fixed scroll compressor body **110**, while the key coupling first keys **144** engage a side portion of the slot **115** to prevent rotations of the key coupling **140**. Limit tabs **111** also provide additional (to limit tabs **113**) axial limit stops.

Though not visible in the view of FIG. **8**, each of the pair of second radially-outward projecting limit tabs **113** (see FIG. **7**) is nested in its respective notched section **166** of the pilot ring **160** to constrain axial movement of the fixed scroll compressor body **110** thereby defining a limit to the available range of axial movement of the fixed scroll compressor body **110**. The pilot ring notched sections **166** are configured to provide some clearance between the pilot ring **160** and the pair of second radially-outward projecting limit tabs **113** to provide for axial restraint between the fixed and movable scroll compressor bodies **110**, **112** during scroll compressor operation. However, the radially-outward projecting limit tabs **113** and notched sections **166** also keep the extent of axial movement of the fixed scroll compressor body **110** to within an acceptable range.

It should be noted that "limit tab" is used generically to refer to either or both of the radially-outward projecting limit tabs **111**, **113**. Embodiments of the invention may include just one of the pairs of the radially-outward projecting limit tabs, or possibly just one radially-outward projecting limit tab, and particular claims herein may encompass these various alternative embodiments

As illustrated in FIG. **8**, the crankcase **42** and pilot ring **160** design allow for the key coupling **140**, and the fixed and movable scroll compressor bodies **110**, **112** to be of a diameter that is approximately equal to that of the crankcase **42** and pilot ring **160**. As shown in FIG. **1**, the diameters of these components may abut or nearly abut the inner surface of the outer housing **12**, and, as such, the diameters of these components are approximately equal to the inner diameter of the outer housing **12**. It is also evident that when the key coupling **140** is as large as the surrounding compressor outer housing **12** allows, this in turn provides more room inside the key coupling **140** for a larger thrust bearing which in turn allows a larger scroll set. This maximizes the scroll compressor **14** displacement available within a given diameter outer housing **12**, and thus uses less material at less cost than in conventional scroll compressor designs.

It is contemplated that the embodiments of FIGS. **7** and **8** in which the first scroll compressor body **110** includes four radially-outward projecting limit tabs **111**, **113**, these limit tabs **111**, **113** could provide radial restraint of the first scroll compressor body **110**, as well as axial and rotation restraint. For example, radially-outward projecting limit tabs **113** could be configured to fit snugly with notched sections **166** such that these limit tabs **113** sufficiently limit radial movement of the first scroll compressor body **110** along first lateral axis **146**. Additionally, each of the radially-outward-projecting limit tabs **111** could have a notched portion configured to abut the portion of the first inner wall **169** adjacent the slots **162** of the pilot ring **160** to provide radial restraint along second lateral axis **154**. While this approach could potentially require maintaining a certain tolerance for the limit tabs **111**, **113** or the notched section **166** and slots **162**, in these instances, there would be no need to precisely tolerance the entire first inner wall **169** of the pilot ring **160**, as this particular feature would not be needed to provide radial restraint of the first scroll compressor body **110**.

With reference to FIGS. **9**-**12**, the upper side (e.g. the side opposite the scroll rib) of the fixed scroll **110** supports a floating seal **170** above which is disposed the separator plate **30**. In the embodiment shown, to accommodate the floating seal **170**, the upper side of the fixed scroll compressor body **110** includes an annular and, more specifically, the cylindrical inner hub region **172**, and the peripheral rim **174** spaced radially outward from the inner hub region **172**. The inner hub region **172** and the peripheral rim **174** are connected by a radially-extending disc region **176** of the base **116**. As shown in FIG. **11**, the underside of the floating seal **170** has circular cutout adapted to accommodate the inner hub region **172** of the fixed scroll compressor body **110**. Further, as can be seen from FIGS. **9** and **10**, the perimeter wall **173** of the floating seal is adapted to fit somewhat snugly inside the peripheral rim **174**. In this manner, the fixed scroll compressor body **110** centers and holds the floating seal **170** with respect to the central axis **54**.

In a particular embodiment of the invention, a central region of the floating seal **170** includes a plurality of openings **175**. In the embodiment shown, one of the plurality of openings **175** is centered on the central axis **54**. That central opening **177** is adapted to receive a rod **181** which is affixed to the floating seal **170**. As shown in FIGS. **9** through **12**, a ring valve **179** is assembled to the floating seal **170** such that the ring valve **179** covers the plurality of openings **175** in the floating seal **170**, except for the central opening **177** through which the rod **181** is inserted. The rod **181** includes an upper flange **183** with a plurality of openings **185** therethrough, and a stem **187**. As can be seen in FIG. **9**, the separator plate **30** has a center hole **33**. The upper flange **183** of rod **181** is adapted to pass through the center hole **33**, while the stem **187** is inserted through central opening **177**. The ring valve **179** slides up and down the rod **181** as needed to prevent back flow from a high-pressure chamber **180**. With this arrangement, the combination of the separator plate **30** and the fixed scroll compressor body **110** serve to separate the high pressure chamber **180** from a lower pressure region **188** within the outer housing **12**. Rod **181** guides and limits the motion of the ring valve **179**. While the separator plate **30** is shown as engaging and constrained radially within the cylindrical side wall region **32** of the top end housing section **26**, the separator plate **30** could alternatively be cylindrically located and axially supported by some portion or component of the scroll compressor **14**.

In certain embodiments, when the floating seal **170** is installed in the space between the inner hub region **172** and the peripheral rim **174**, the space beneath the floating seal **170** is pressurized by a vent hole (not shown) drilled through the fixed scroll compressor body **110** to chamber **122** (shown in FIG. **2**). This pushes the floating seal **170** up against the separator plate **30** (shown in FIG. **9**). A circular rib **182** presses against the underside of the separator plate **30** forming a seal between high-pressure discharge gas and low-pressure suction gas.

While the separator plate **30** could be a stamped steel component, it could also be constructed as a cast and/or machined member (and may be made from steel or aluminum) to provide the ability and structural features necessary to operate in proximity to the high-pressure refrigerant gases output by the scroll compressor **14**. By casting or machining the separator plate **30** in this manner, heavy stamping of such components can be avoided.

During operation, the scroll compressor assembly **10** is operable to receive low-pressure refrigerant at the housing inlet port **18** and compress the refrigerant for delivery to the high-pressure chamber **180** where it can be output through the housing outlet port **20**. This allows the low-pressure refrigerant to flow across the electrical motor assembly **40** and thereby cool and carry away from the electrical motor assembly **40** heat which can be generated by operation of the motor. Low-pressure refrigerant can then pass longitudinally through the electrical motor assembly **40**, around and through void spaces therein toward the scroll compressor **14**. The low-pressure refrigerant fills the chamber **31** formed between the electrical motor assembly **40** and the outer housing **12**. From the chamber **31**, the low-pressure refrigerant can pass through the upper bearing member or crankcase **42** through the plurality of spaces **244** that are defined by recesses around the circumference of the crankcase **42** in order to create gaps between the crankcase **42** and the outer housing **12**. The plurality of spaces **244** may be angularly spaced relative to the circumference of the crankcase **42**.

After passing through the plurality of spaces **244** in the crankcase **42**, the low-pressure refrigerant then enters the intake area **124** between the fixed and movable scroll compressor bodies **110**, **112**. From the intake area **124**, the low-pressure refrigerant enters between the scroll ribs **114**, **118** on opposite sides (one intake on each side of the fixed scroll compressor body **110**) and is progressively compressed through chambers **122** until the refrigerant reaches its maximum compressed state at the compression outlet **126** from which it subsequently passes through the floating seal **170** via the plurality of openings **175** and into the high-pressure chamber **180**. From this high-pressure chamber **180**, high-pressure compressed refrigerant then flows from the scroll compressor assembly **10** through the housing outlet port **20**.

FIGS. **13** and **14** illustrate an alternate embodiment of the invention. Instead of a crankcase **42** formed as a single piece, FIGS. **13** and **14** show an upper bearing member or crankcase **199** combined with a separate collar member **198**, which provides axial thrust support for the scroll compressor **14**. In a particular embodiment, the collar member **198** is assembled into the upper portion of the upper bearing member or crankcase **199** along stepped annular interface **100**. Having a separate collar member **198** allows for a counterweight **230** to be assembled within the crankcase **199**, which is attached to the pilot ring **160**. This allows for a more compact assembly than described in the previous embodiment where the counterweight **130** was located outside of the crankcase **42**.

As is evident from the exploded view of FIG. **13** and as stated above, the pilot ring **160** can be attached to the upper bearing member or crankcase **199** via a plurality of threaded fasteners to the upper bearing member **199** in the same manner that it was attached to crankcase **42** in the previous embodiment. The flattened profile of the counterweight **230** allows for it to be nested within an interior portion **201** of the upper bearing member **199** without interfering with the collar member **198**, the key coupling **140**, or the movable scroll compressor body **112**.

All references, including publications, patent applications, and patents cited herein are hereby incorporated by reference to the same extent as if each reference were individually and specifically indicated to be incorporated by reference and were set forth in its entirety herein.

For purposes of this disclosure, the term "coupled" means the joining of two components (electrical or mechanical) directly or indirectly to one another. Such joining may be stationary in nature or moveable in nature. Such joining may be achieved with the two components (electrical or mechanical) and any additional intermediate members being integrally formed as a single unitary body with one another or the two components and any additional member being attached to one another. Such adjoining may be permanent in nature or alternatively be removable or releasable in nature.

The use of the terms "a" and "an" and "the" and similar referents in the context of describing the embodiments (especially in the context of the following claims) is to be construed to cover both the singular and the plural, unless otherwise indicated herein or clearly contradicted by context. The terms "comprising," "having," "including," and "containing" are to be construed as open-ended terms (i.e., meaning "including, but not limited to,") unless otherwise noted. Recitation of ranges of values herein are merely intended to serve as a shorthand method of referring individually to each separate value falling within the range, unless otherwise indicated herein, and each separate value is incorporated into the specification as if it were individually recited herein. All methods described herein can be performed in any suitable order unless otherwise indicated herein or otherwise clearly contradicted by context. The use of any and all examples, or exemplary language (e.g., "such as") provided herein, is intended merely to better illuminate the invention and does not pose a limitation on the scope of the invention unless otherwise claimed. No language in the specification should be construed as indicating any non-claimed element as essential.

Preferred embodiments are described herein, including the best mode known to the inventors for carrying out the invention. Variations of those preferred embodiments may become apparent to those of ordinary skill in the art upon reading the foregoing description. The inventors expect skilled artisans to employ such variations as appropriate, and the inventors intend for the invention to be practiced otherwise than as specifically described herein. Accordingly, this disclosure includes all modifications and equivalents of the subject matter recited in the claims appended hereto as permitted by applicable law. Moreover, any combination of the above-described elements in all possible variations thereof is encompassed by the disclosure unless otherwise indicated herein or otherwise clearly contradicted by context.

What is claimed is:

**1**. A scroll compressor comprising:

a housing having an upper end and a lower end;

a pair of scroll compressor bodies disposed in the housing, the scroll bodies including a first scroll body and a second scroll body, the first and second scroll bodies having respective bases and respective scroll ribs that project from the respective bases, wherein the scroll ribs mutually engage, the second scroll body being movable relative to the first scroll body for compressing fluid;

a pilot ring that engages a perimeter surface of the first scroll body to limit movement of the first scroll body in the radial direction, the first scroll body having a first radially-outward-projecting limit tab being configured to limit movement of the first scroll body in at least one of the axial and rotational directions;

a stationary lower bearing member disposed proximate the lower end of the housing;

a motor disposed in the housing, with the motor including a stator and a rotor with the rotor coupled to a shaft configured to rotate within the housing and with one of the pair of scroll compressor bodies coupled to the shaft; and

a load transfer apparatus comprising;

a cylindrical central hub defined by the stationary lower bearing member, wherein the central hub defines an opening in the lower bearing member;

a cylindrical bearing configured to seat in the opening, with the cylindrical bearing further configured to receive one end of the shaft; and

a thrust washer disposed in the opening of the central hub and captured axially within the lower bearing member by the cylindrical bearing the lower bearing member and central hub being positioned between the motor and an oil sump at the lower end of the housing,

wherein an axial load along the centerline of the shaft transmits to the stationary lower bearing member through the thrust washer.

**2**. The scroll compressor of claim **1**, wherein the thrust washer is fixed axially and rotationally in the opening without a fastener or an adhesive.

**3**. The scroll compressor of claim **1** wherein the thrust washer includes a smooth circumference.

**4**. The scroll compressor of claim **1**, wherein the thrust washer is metal.

**5**. The scroll compressor of claim **1**, wherein the cylindrical bearing is composed of a matrix of metal and a polymeric layer.

**6**. The scroll compressor of claim **5**, wherein the cylindrical bearing is lubricated by oil transferring through an orifice defined in the shaft.

**7**. The scroll compressor of claim **1**, wherein the pilot ring is formed separately from a crankcase, the pilot ring being attached to the crankcase via a plurality of posts extending axially therebetween, the first and second scroll bodies being disposed within the attached pilot ring and crankcase, and further comprising a key coupling that acts upon the second scroll body, the key coupling being disposed within the attached pilot ring and crankcase, and extending into spaces between adjacent posts, and whereby the spaces allow the pilot ring, crankcase, and key coupling to have outer diameters that are approximately equal to the inner diameter of the housing.

**8**. The scroll compressor of claim **1**, wherein the thrust washer is frustoconical.

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